

### A NEW SPEED INDICATOR FOR MARINE PROPELLERS.

When the steamship "Perry G. Walker" collided with the lock gates at Sault Sainte Marie, causing the wreck of two other steamers and doing damage to the locks which required weeks for repair, the captain stated under examination that he had signaled to his engineer to go astern, but that his signal had somehow been misunderstood, and the engines had started full-speed ahead. Such an accident is conclusive and incontrovertible evidence of the need of a reliable system of indicating the direction and speed of rotation of the propellers of vessels. It is but one instance of a chapter of marine accidents occurring annually from either the incorrect interpretation of signals given to the engine room from the bridge or the execution of signals given by the bridge which, owing to mental stress from impending accident, are incorrectly given. It is always extremely difficult for a board of inquiry to determine just with whom the error lies.

In the above instance, it is claimed by the captain that the correct signals were given, but instead of the engines being reversed at the critical moment, they were sent ahead; and before the error was discovered, such headway had been gathered by the vessel as to preclude all hope of stopping her within the limited lock space.

There is no question of the importance of enabling the captain and pilot to be at all times familiar with the interpretation and execution of signals. Errors are thereby immediately discernible, and correspondingly corrected before damage is done.

In the absence of a tachometer to show at a glance the rate in revolutions per minute at which the propeller shaft is turning, signals are executed by the engineer according to his best judgment. For instance, the execution of "half speed astern" may vary eight or ten revolutions per minute, and the pilot, depending upon a speed-checking effect, may be thrown off in his calculations by too slow a rate of turning of the engines.

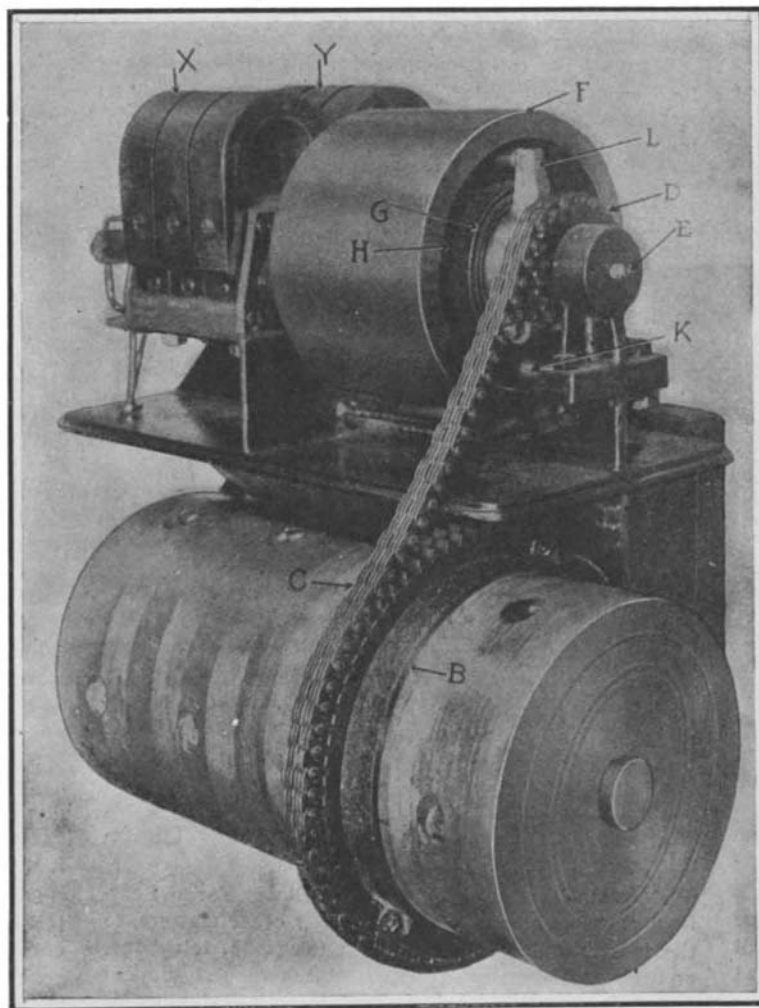
When equipped with a tachometer system, however, the signals can be obeyed at an exact predetermined propeller-shaft speed, with corresponding increased accuracy and efficiency of handling the vessel. Warships in line or column formation must correctly execute the orders of the flagship, setting their speed to conform to the desired headway between ships, quickly and accurately. Otherwise a collision is probable.

The absence of an accurate and dependable tachometer up to the present, has made it necessary to arrive at the revolutions per minute by noting the turns successively by the revolution counter for preferably at least a half minute. If the speed of the shaft is too high, a rough guess must be made as to how much the throttle is to be closed, and another counting gone through. All this takes time, and is on too much of a cut-and-try system. With a tachometer to guide him, the man at the throttle has but to operate the throttle until the pointer of the tachometer rests on the desired R. P. M.

Range finding, for the accurate sighting of the guns, includes the determination of the distance of the object to be fired at, angle at which the warship is

approaching or receding from the target, and the speed at which the vessel is traveling.

The first two factors are quickly and accurately determined by means of the modern range finder in the hands of skilled men, located on the masts or range towers of the warship. This is telephoned to the fire control sub-station. It then becomes imperative that the rate at which the engines are turning over at that instant be immediately determined, in order that the



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proper instructions may be telephoned at once to the turrets. The sooner the discharge of the projectile is effected after the range has been determined, the more accurate is the aim, and the greater the execution done.

In these calculations, the effect on ship speed by propeller speed, taking into consideration the extent and direction of wind and tide, is quickly and accurately calculated.

Relation between ship speed and propeller speed is frequently calibrated with due reference to increased fouling of the ship's bottom from marine growth, and is immediately available. Even when the engine-room forces are endeavoring to maintain an exact pre-arranged speed of rotation, this speed often varies, owing to the absence of accurate deadbeat tachometers for indicating at all times the rate of revolution.

Aside from the strategic advantages of a tachometer for indicating engine speed of rotation, the economic

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### POWERFUL HOISTING AND CONVEYING MACHINE.

Nowhere in the field of mechanical engineering has American ingenuity in the design of labor-saving plants been shown to more striking effect than those great hoisting and conveying plants, which are such a prominent factor in our modern constructive and industrial operations. The rapid and cheap raising, removal, distribution and deposit of materials in large bulk is one of the most serious problems of the day; and it is the ingenious solution offered by American hoisting and conveying apparatus that has enabled our engineers to dig canals, build embankments, handle enormous loads of coal, iron ore, wheat, and corn with an economy undreamed of in an earlier day. We present illustrations of a powerful electric bridge tramway, designed and erected by the Brown Hoisting and Machinery Company for the Michigan Alkali Company, which is an excellent sample of the type of machinery above referred to.

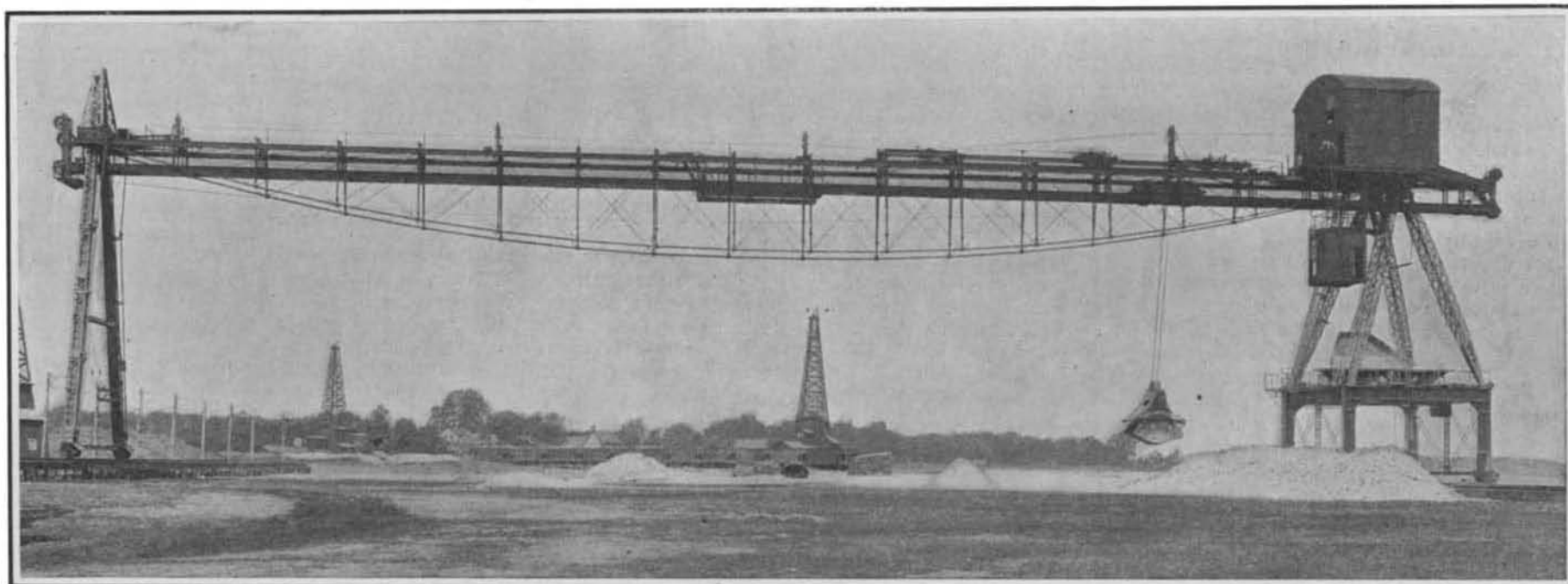
The bridge, which is designed to handle the limestone in the stock yard of the company, has a span of 256 feet from center of pier to center of shear, with the center depth of 17 feet, and the total over-all of the structure is 286 feet 4½ inches. The height from top of rail to top of bridge at the shear is 59 feet 9 inches, and at the pier 61 feet 9 inches, the bridge being level. To the bridge span and its projection is attached a runway carrying a special trolley, arranged to handle either a two-rope grab bucket or a scraper bucket.

The pier consists of two specially designed shear legs mounted on a portal structure, arranged to straddle over two lines of railway track. The two shears are joined together at the top by a yoke connection, designed to carry the bridge structure. By this arrangement a free opening is allowed for the passage of the buckets through the pier support. The structure of the pier throughout is of medium open-hearth steel. The portal or lower portion of the pier consists of two pairs of legs joined together by girders and braces, and arranged to carry a bin for the reception and distribution of the limestone. The lower portion of the portal is mounted on four two-wheel equalizing trucks. These wheels are con-

nected by bevel and spur gears to the driving machinery in the house on the bridge. The shear-leg support is of A-frame construction, mounted on two-wheel equalizing trucks, arranged to run on a single line of rail. At the top of the shear is a ball casting, upon which the main bridge is hung. The track wheels are connected with the moving gear in the engine by bevel and spur gears.

The bridge span consists of two parabolic pin-connected trusses, supporting the cross beams, from which the track stringers are suspended. The bridge span is supported on the pier support by roller bearings, and held in place by a vertical center pin. At the shear support it is hung from a ball-and-socket connection, in such a manner that the bridge may be skewed in either direction from its normal axis, so as to give an angle of one foot crosswise to nine feet lengthwise of the bridge span. The moving gear is operated from the main operating mechanism located

(Continued on page 169.)



Length over all, 286 feet 4½ inches; depth of trusses, 17 feet; height of bridge above ground, 61 feet 9 inches.

Hoisting and conveying machine; capacity 200 tons per hour.

POWERFUL ELECTRIC HOISTING AND CONVEYING MACHINERY.

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 Pump, piston, H. P. Wedin ..... 931,535  
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 Rail joint, T. K. Diehl ..... 931,577  
 Rail joint, Frederick & Chapman ..... 931,589  
 Rail joint truss support, L. Mueller ..... 931,459  
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 Railway frog, B. J. Morgan ..... 931,073  
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 Tie plate, A. L. Stanford ..... 931,517  
 Time recorder, W. Beresford ..... 931,254  
 Tire antiskidding device, T. I. Duffy ..... 931,284

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of magnetos supplying all the indicators, which are connected in multiple.

A very important fact in Mr. Hutchison's system is the use of an exceedingly small current value and low voltage; hence, in the event of the indicators being located in proximity to ammunition, should the circuit be opened by the breaking of a wire or otherwise, the resultant spark is barely perceptible and cool, therefore incapable of igniting anything. Should the line become short-circuited, no heating effect whatever is produced and no damage done to the magnetos, as they can run for an indefinite time on dead short circuit. No effect is produced on the compass by the current in the wires, and, taken all in all, the system seems to meet every requirement of accuracy and safety.

POWERFUL HOISTING AND CONVEYING MACHINE.

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in the machinery house at the pier end of the bridge, the power from which is transferred to the moving gear wheels by a line of shafting on top of the bridge, thence to lines of shafting down the pier and shear supports, and thence by a proper train of gears to the wheels.

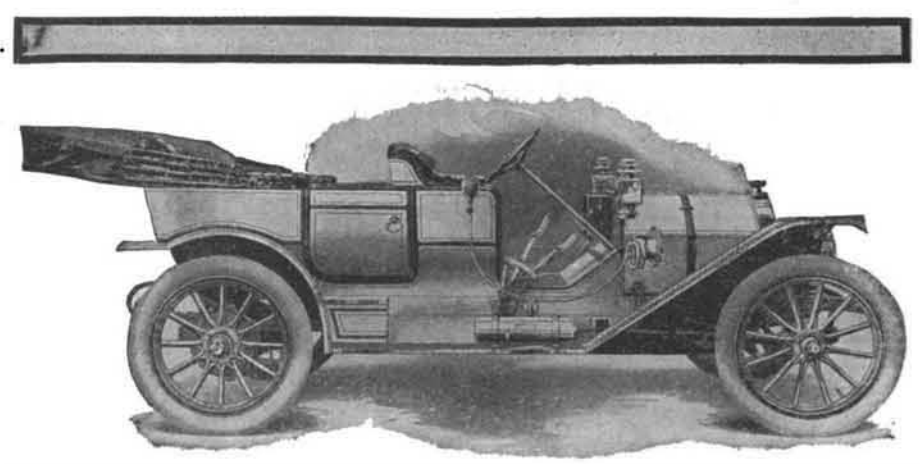
The moving gear mechanism is attached to the operating mechanism by means of friction clutches, so arranged that the motion of the crane may be made in either direction at the will of the operator. Further provision is made for disconnecting the gearing, in order to skew the bridge within the limits given.

The main operating mechanism for the bridge is located in the machinery house, and consists of two drums. Each of these drums is mounted loose on its supporting shaft. Each drum is controlled by a Brown friction clutch and a band friction brake. The drums are connected one to the other through an equalizing gear mechanism, which is equipped also with a powerful foot brake, so that the two main operating drums may be made to rotate in opposite directions with the same speed. This feature of connecting the two drums is one of the all-important points in the Brown two-drum operating mechanism. The main operating machinery is further arranged to operate from the intermediate shaft the bridge crane moving gear mechanism, to which the supporting truck wheels are connected by shafting and gearing. This mechanism is controlled also by a powerful clutch and band friction foot brake.

In further connection with the two main operating drums is a small closing drum which, in conjunction with the two main drums, controls all of the motions of hoisting and lowering the load and traveling the trolley. The trolley, which in reality forms also a part of the main hoisting mechanism, is specially designed to operate in conjunction with the drum arrangement above described. In general, this trolley consists of a steel structural frame mounted on four turned cast-steel track wheels arranged to run on the trolley runway of the bridge crane. In the trolley there are mounted specially-designed drums, which run loose on their supporting shafts. The large section of each drum on the trolley is connected to the main hoisting drums in the machinery house in the following manner:

One length of wire rope will connect, from the under side, one of the main hoisting drums to the top side of one of the large sections of the trolley drums. The under side of this trolley drum is connected by wire ropes to the top side of the other main hoisting drum. The top side of this same hoisting drum is connected by suitable cable to the top side of the other trolley drum, and likewise this trolley drum from the under side is connected to the underside of the other main hoisting drum. By this arrangement of ropes, if one of the main

(Continued on page 170.)



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hoisting drums is connected through its clutch mechanism to the driving shaft, and the other main hoisting drum is allowed to work through the equalizing gear mechanism loose on its supporting shaft...

In the operation, however, of the scoop bucket, with which the above bridge crane is equipped, only one main drum on the trolley is used, this being the drum operating the shell lines of the drum operating the shell lines of the grab bucket...

The 200 horse-power motor and electrical equipment is designed so that during

(Concluded on page 171.)

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ing a run of twenty-four hours, when handling material at the speeds as specified in the contract, no part of the motor will rise in temperature more than 70 deg. C. above the surrounding air. All electrical equipment is designed for a direct current of 220 volts. The grab bucket has a cubical capacity of 100 cubic feet of limestone, and the scoop bucket of 132 cubic feet. Both buckets are especially designed for working in limestone. The grab bucket has an overall width of about 7 feet 6 inches and an over-all length when open of 17 feet 6 inches. The capacity of the machine is 200 tons per hour. The hoisting speed is 250 to 275 feet per minute; the racking speed 900 feet per minute; and the whole bridge travels at the rate of 100 to 150 feet per minute.

MEASURING A RIVER'S FLOW.

(Concluded from page 160.)

the velocity recorded at a number of different depths in each strip. By the vertical integration method the meter is moved at a slow uniform speed from the surface of the stream to the bottom and back again.

For convenience of reference and comparison the results obtained are plotted in the form of a curve on a chart.

Another illustration shows the Great Falls of the Missouri River in Montana. A gaging station at the point from which the photograph is taken was established by the Geological Survey in July, 1902.

The river is favorable at this point for water-power development and shows the kind of stream, apart from navigable rivers, measured and reported upon by the Survey. In this way the Survey constantly brings to the attention of the investing and developing public many previously unnoticed but valuable water-power sites.

We are indebted to the director of the U. S. Geological Survey for the use of the accompanying illustrations.

THE AVIATION MEETING AT RHEIMS.

(Concluded from page 159.)

with their Bleriot monoplanes. Both Bleriot and Curtiss tried to lower their speed records for one circuit of the course, and the latter succeeded in making 2 seconds better time than before. His time of 8:09 1/5 corresponds to almost 45.7 miles an hour. Bleriot made the circuit in 8:08 2/5, which was 4 seconds slower than formerly.

At the end of 2 hours, 22 minutes, and 51 seconds, Farman had flown 140 kilometers (86.99 miles) and beaten Paulhan's record. It was getting dark rapidly and the spectators could only see the machine as it passed before the grand stand. Ten minutes and 19 seconds later he completed his fifteenth round, and less than five minutes later he had beaten Latham's record. One hundred and sixty kilometers (99.4 miles) were covered in 2:43:35 2/5, and 180 kilometers (111.83 miles) in 3 hours, 4 minutes, 55 2/5 seconds. As it was now 7:30, the nineteenth round afterward made by Farman was not counted in the official figures. He actually covered over 190 kilometers (118.06 miles) and remained in the air all told about 3 1/4 hours. As he finished in front of the grand stand a searchlight was thrown upon him. He was pulled from his machine and carried upon the shoulders of his friends, receiving a decided ovation. Thus, for the second time, he has won a \$10,000 cash prize, the first instance being when he flew 1 kilometer in a closed circuit on January 13th, 1908. It is possible that he will try again to win this sum by making the 140-mile flight from New York to Albany. In the flight for the Grand Prix, he carried enough fuel to fly 3 1/2 hours.

The other prizes awarded in the Grand Prix de la Champagne distance race were as follows:

Second, \$5,000, won by Hubert Latham on his Antoinette monoplane. Distance, 154.5 kilometers (96 miles).

Third, \$2,000, won by M. Paulhan with his Voisin biplane. Distance, 131 kilometers (81.4 miles).

Fourth, \$1,000, won by Count de Lambert with his Wright biplane. Distance, 116 kilometers (72.1 miles).

Fifth, \$1,000, won by Paul M. Tissandier. Distance, 111 kilometers (68.97 miles).

Sixth, \$1,000, won by M. Roger Sommer with a Farman biplane. Distance, 60 kilometers (37.3 miles).

The distances covered by the other competitors were: 50 kilometers (31.1 miles) by M. Delagrang, with a Bleriot monoplane; 40 kilometers (24.9 miles) covered by M. Bleriot with one of his monoplanes; 30 kilometers (18.64 miles) covered by Mr. Curtiss with his biplane; and 21 kilometers (13.04 miles) covered by M. LeFebvre with his Wright machine.

This first aviation meeting has demonstrated beyond a doubt that the real flying machine is here. That aeroplane races will soon supersede the dangerous automobile races, there can be no question. We expect in our next issue to give full details of the successful machines at Rheims and their motors, as well as further particulars of the flights which were accomplished.

Table listing various inventions and their inventors with corresponding page numbers. Includes items like 'Tire cover and fastening therefor', 'Tobacco cutter', 'Track laying machine', etc.

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